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(54) **MOUNTING APPARATUS FOR  
DYNAMICALLY LOADED STRUCTURAL  
JOINTS**

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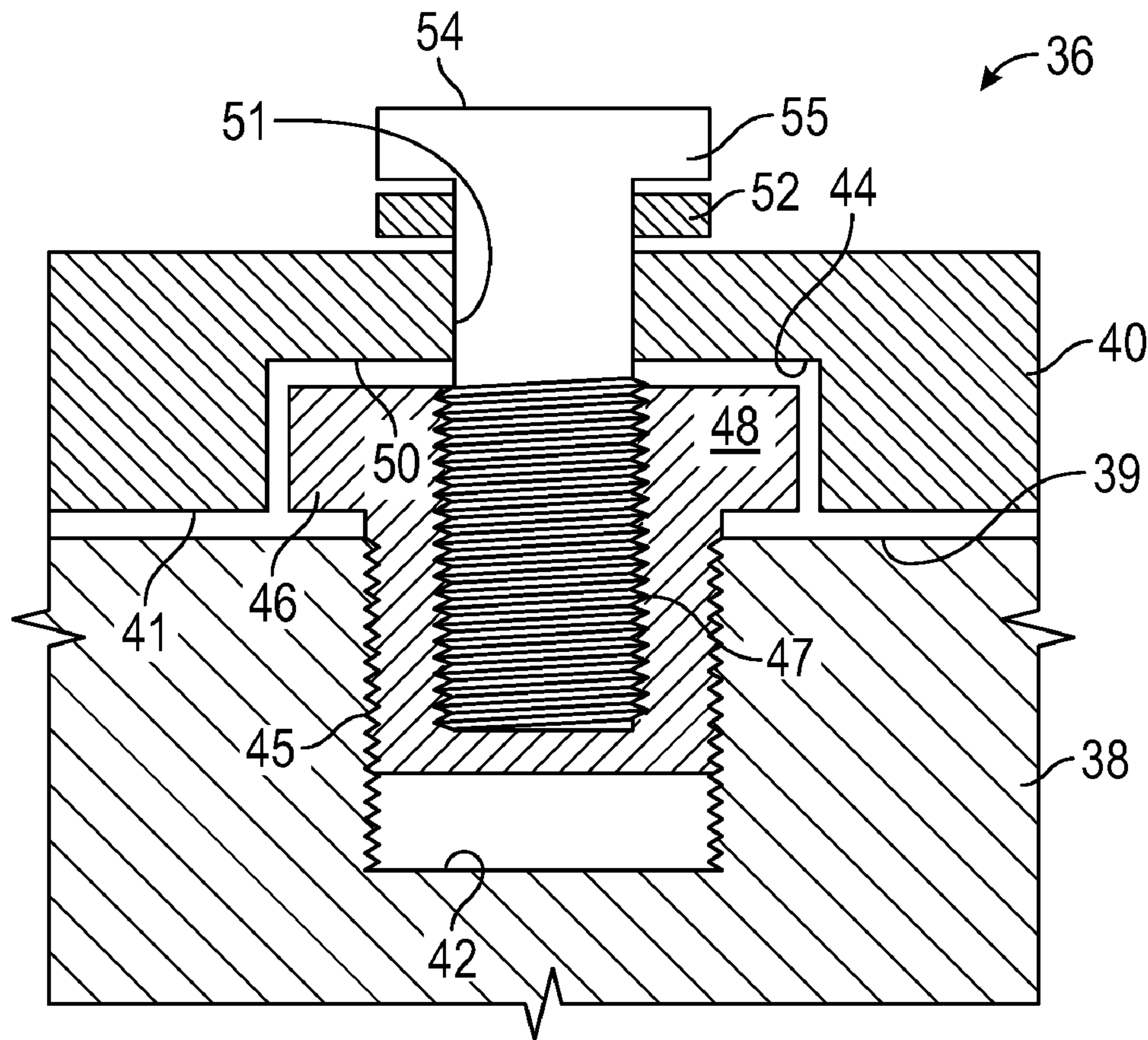
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(57) **ABSTRACT**

A mounting structure apparatus for dynamically loaded structure joints is disclosed. The apparatus includes a first member having an aperture through a first surface and communicating with a counterbore in a second surface. A second member is included having a blind bore coaxial with the first member aperture. An insert is engageable with the counter bore, and a fastener is disclosed insertable into the first member aperture and interactive with the second member to join the first member to the second member.

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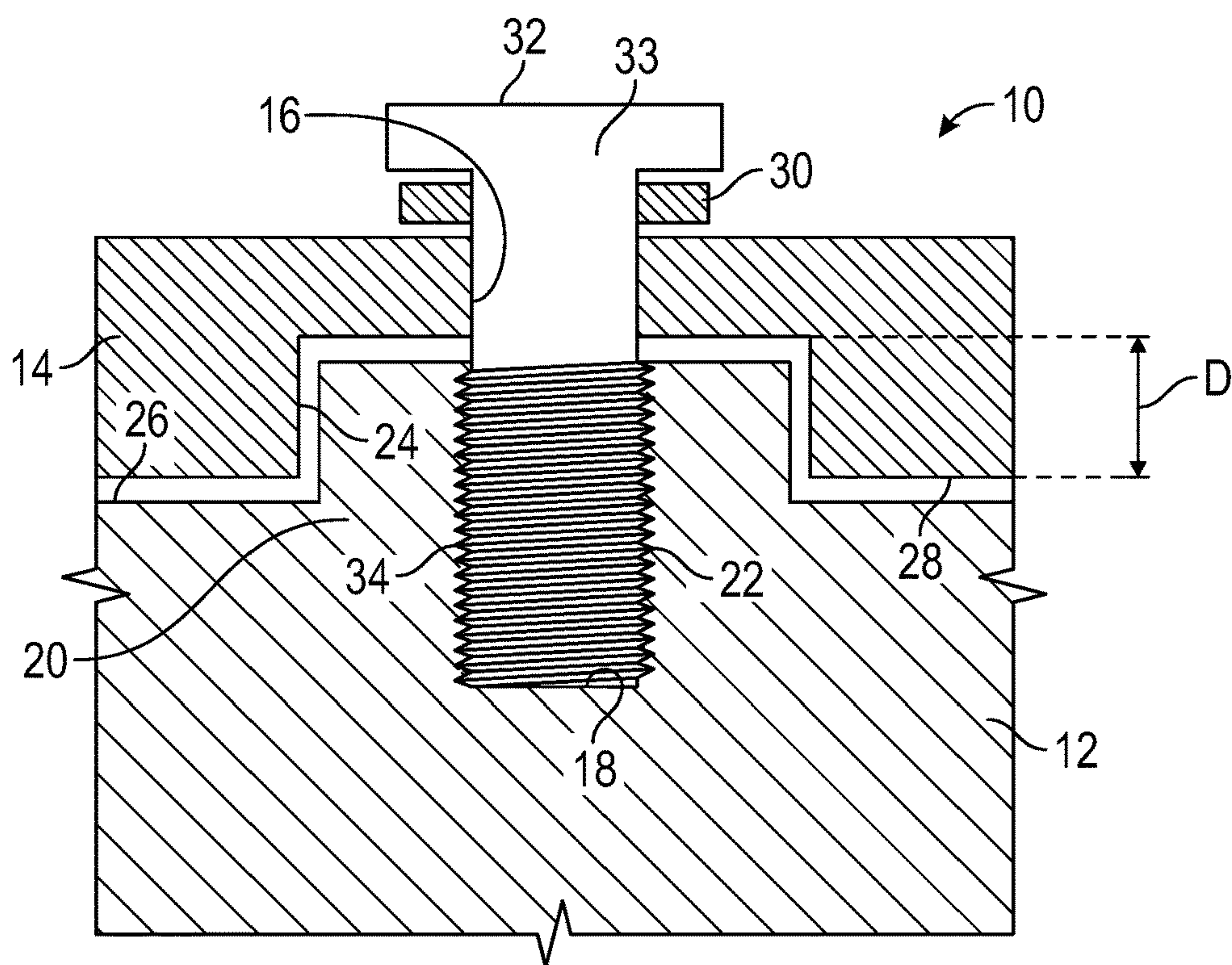


FIG. 1

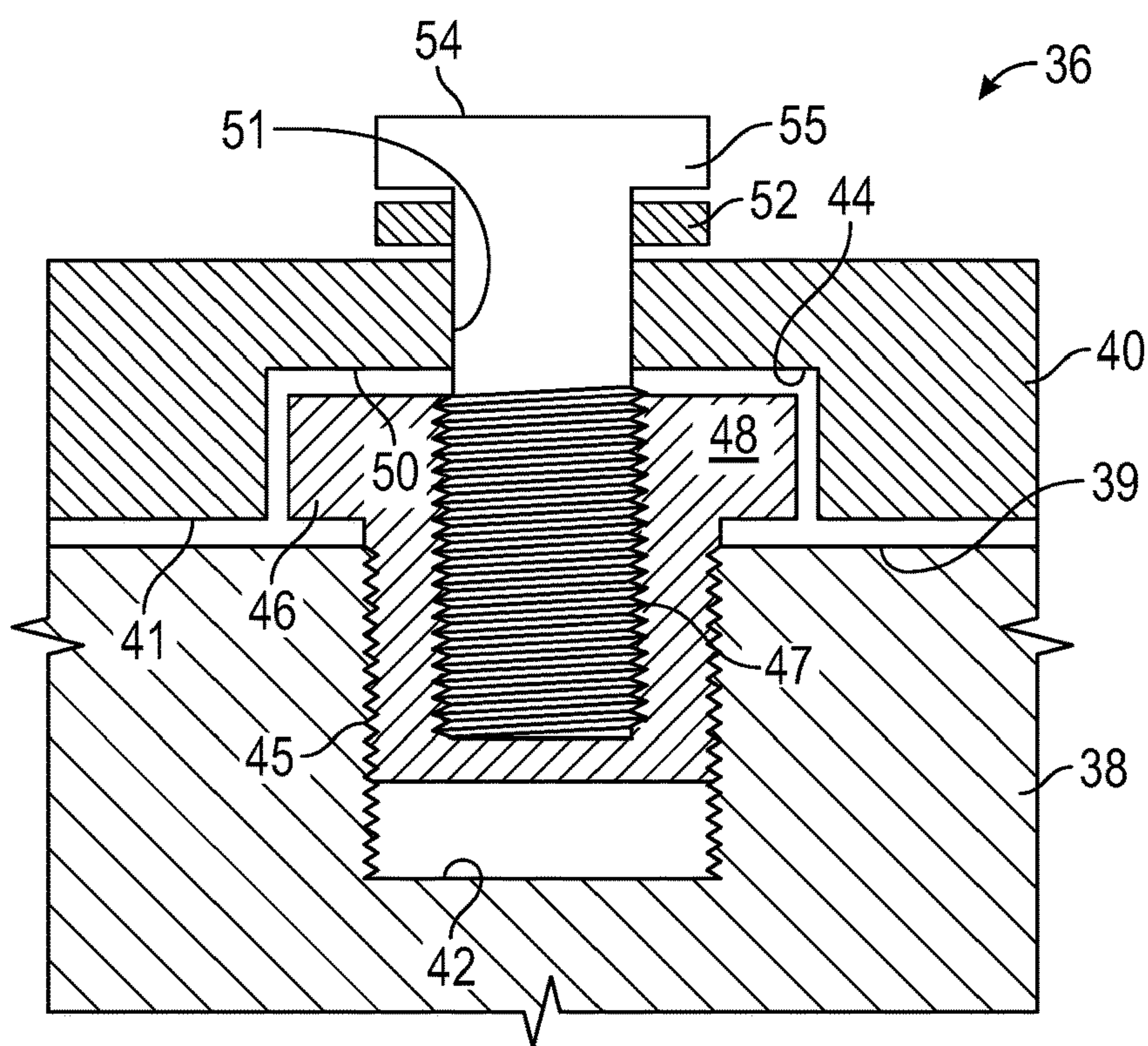
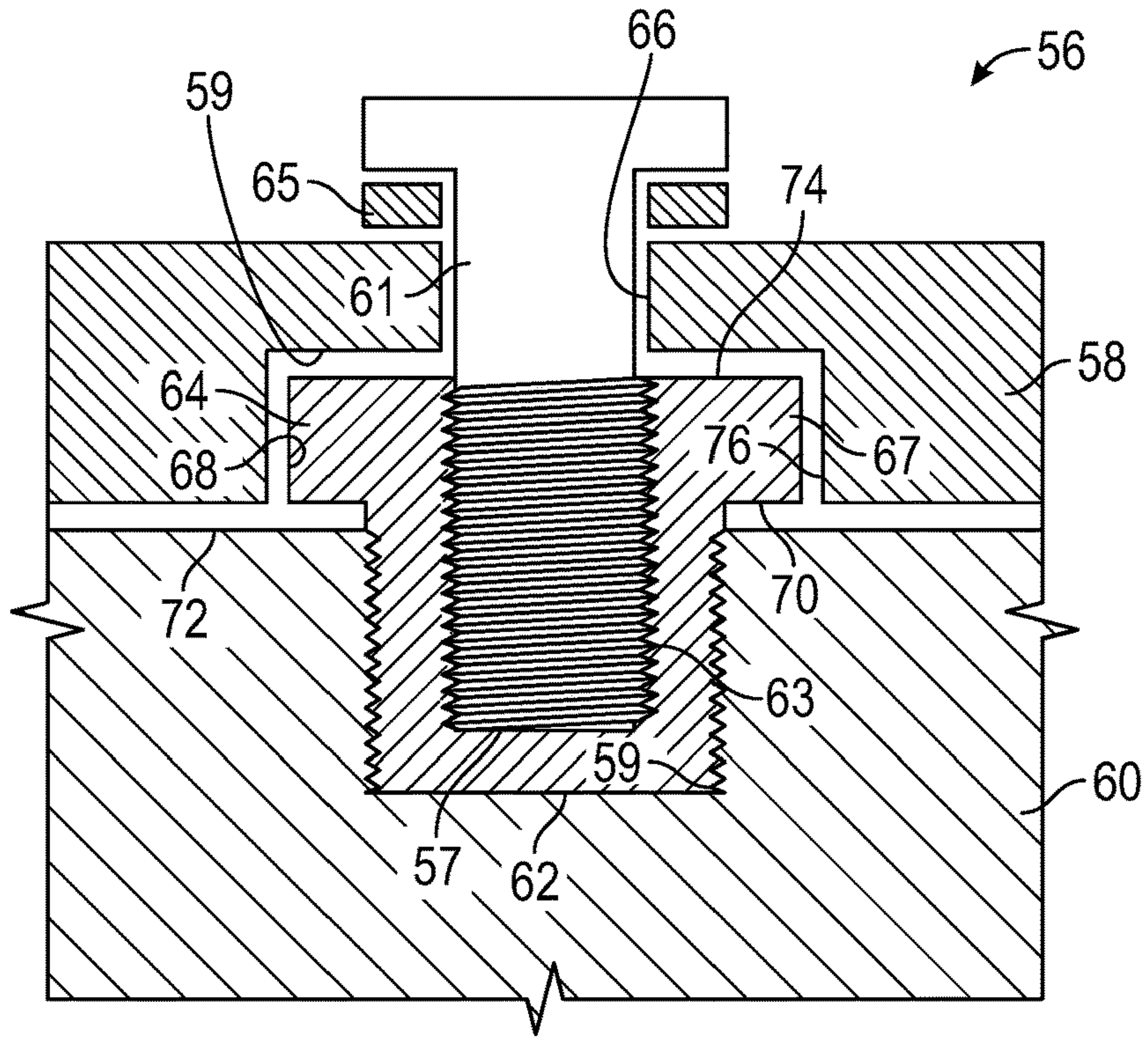
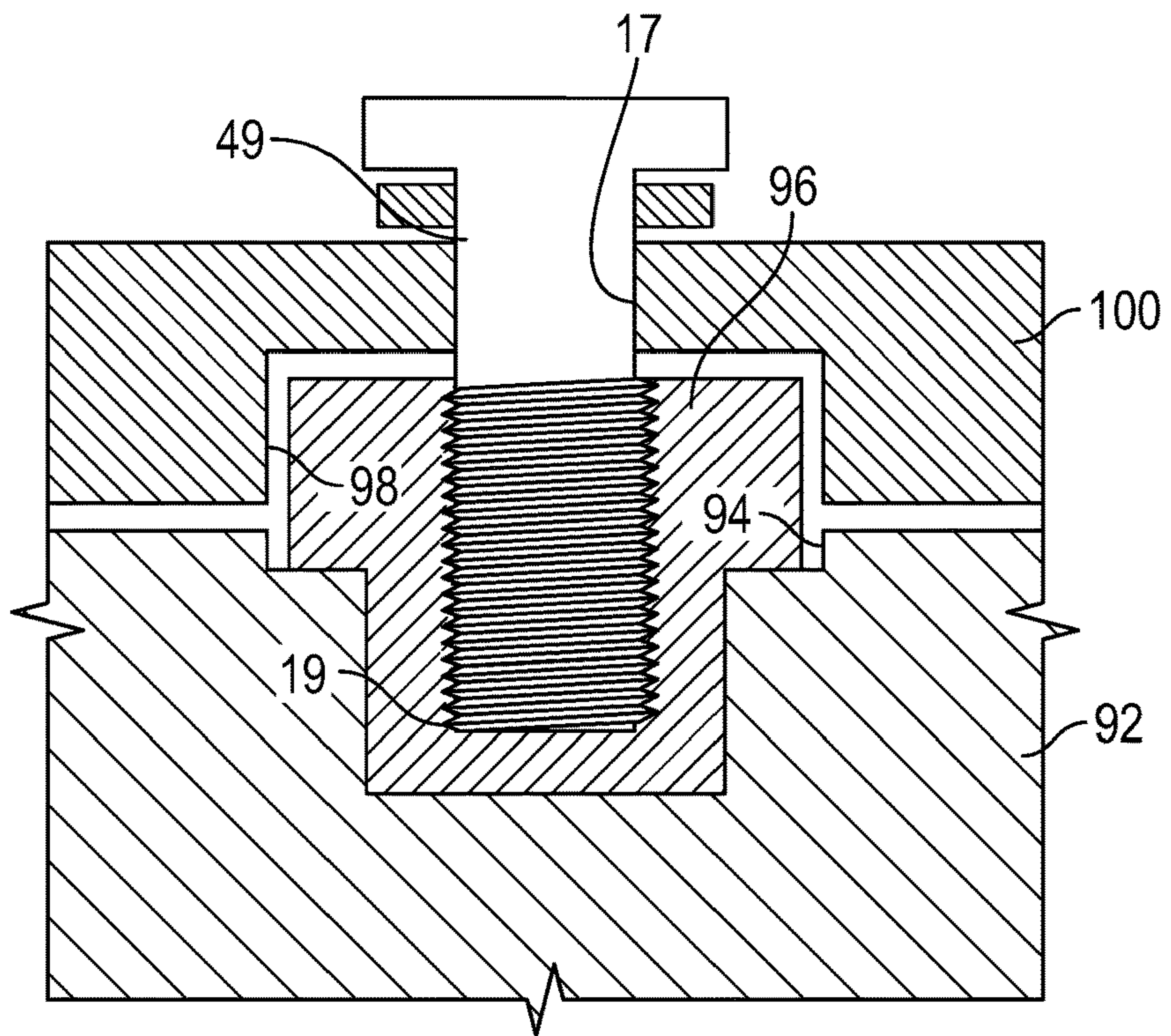


FIG. 2





**FIG. 3**



**FIG. 4**

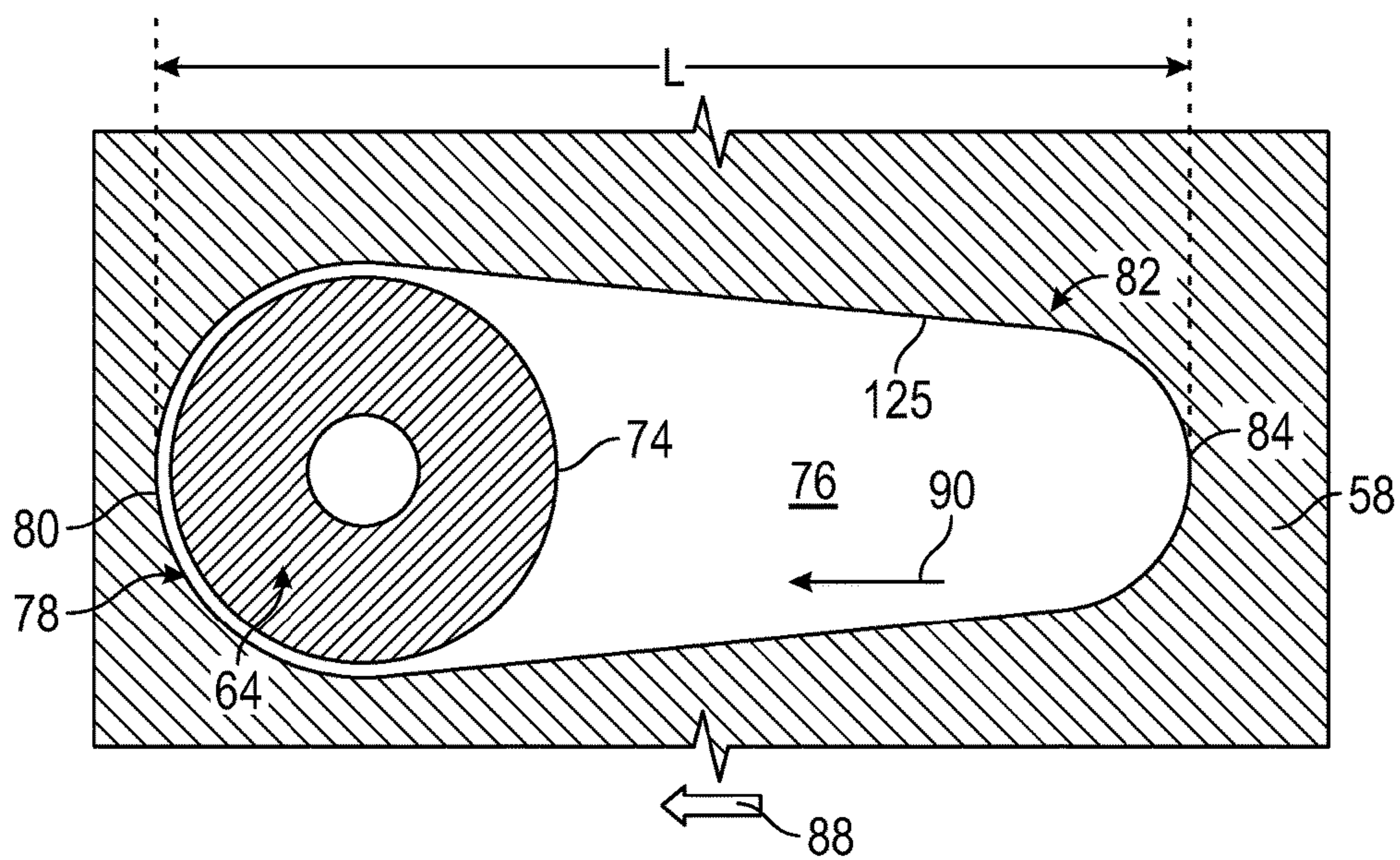


FIG. 5

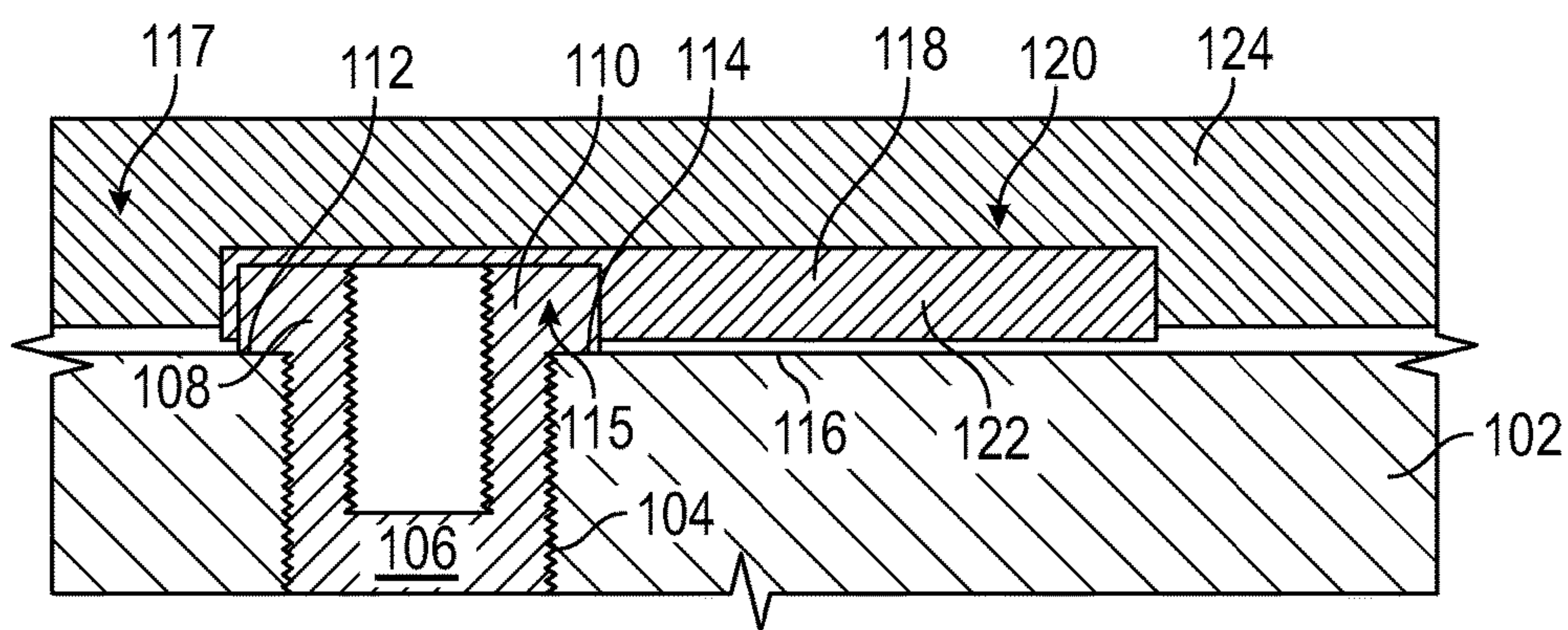


FIG. 6



## MOUNTING APPARATUS FOR DYNAMICALLY LOADED STRUCTURAL JOINTS

### STATEMENT OF GOVERNMENT INTEREST

[0001] This disclosure was made in part with Government support by The United States Department of the Army. The Government has certain rights in the disclosure.

### TECHNICAL FIELD

[0002] This disclosure relates to improved mounting structures for dynamically loaded structural joints. The mounting structure allows shear strength and tensile strength of a joint to be independently tuned by varying sizes of an engaged counter bore and a fastener.

### BACKGROUND

[0003] Existing fasteners are limited in creating a joint with improved shear and tensile strength. Traditional joints consisting of a bolt threaded into a nut are limited in the amount of energy they can absorb prior to failure. This is especially important in situations with dynamically loaded structural joints. Dynamically loaded structural joints, such as those formed when jointing armor plating, are subjected to shear and tensile forces. Existing fasteners can also result in a “secondary projectile” when subjected to shear forces that overcome the fastener strength. It is important to be able to accommodate these forces in armored vehicles which may be subjected to blast forces that may overcome existing fasteners.

[0004] Improvements to such joints are given in the several embodiments disclosed. In one embodiment, one piece of a structure may be attached to another piece of a structure with a bolt and an insert that is mechanically loaded as a result of translational movement of the two pieces. The insert may be threaded into the main piece of structure, and the insert may have internal threads for attachment, and a shoulder feature that sits on the surface of the main structure. The mating piece of the structure has a counterbore that fits over the shoulder of the insert and transfers load when translation occurs between the two structures.

### BRIEF DESCRIPTION OF THE DRAWINGS

[0005] FIG. 1 is a cut away side view of one embodiment of the mounting apparatus;

[0006] FIG. 2 is a cut away side view of another embodiment of the mounting apparatus;

[0007] FIG. 3 is a cut away side view of another embodiment of the mounting apparatus;

[0008] FIG. 4 is a cut away side view of another embodiment of the mounting apparatus;

[0009] FIG. 5 is a cutaway top view of another embodiment of the mounting apparatus showing a the tapered insert within a tapered counterbore in one piece of the apparatus

[0010] FIG. 6 is a cutaway side view of another embodiment of the mounting apparatus with a tapered counterbore.

### DETAILED DESCRIPTION

[0011] All figures and examples herein are intended to be non-limiting; they are mere exemplary iterations and/or embodiments of the claims appended to the end of this

description. Modifications to structure, materials, the order of steps in procedures, etc., are contemplated.

[0012] Referring now to the drawings, and particularly to FIG. 1, there is shown a cutaway side view of one embodiment of the mounting apparatus. Specifically, mounting structure 10 is shown joining first plate 12 and second plate 14. Plate 14 is equipped with an aperture 16 extending there through, and oriented to be coaxial and overlie blind bore 18 in plate 12. Member or plate 12 has insert projection 20 which may be threaded with an internal thread 22 in the bore. Plate 14 has a circular counter bore 24, so that the plate 14 may be moved into engagement with the insert projection 20 so as to engage shoulder 26 on plate 12 to shoulder 28 on plate 14. The counterbore is designed to have a depth D to accommodate at least a portion of the projection 20. A fastener 32, which may be threaded 34, and a washer 30, are used to secure the plates together. Specifically, in this embodiment, plate 14 is placed adjacent to plate 12, and projection 20 extends into the counterbore 24, and the plate 14 is moved toward plate 12 until shoulder 28 on plate 14 engages shoulder 26 on plate 12. Threaded fastener 32 is passed through washer 30 and is inserted through aperture 16 until it threadably engages threads 22 in the blind bore 18. The fastener is threadably inserted into threaded engagement until the head 33 of the fastener is in engagement with the washer, thereby securing the plate in place. The mounting structure as described imparts improved structural integrity to dynamically loaded joints.

[0013] FIG. 2 is a different embodiment of the mounting apparatus. Specifically, mounting structure 36 is shown joining two plates 38 and 40. Plate 38 is equipped with a blind bore 42 and plate 40 is equipped with a circular counterbore 44. An insert 46 having external threads 45 is threaded into engagement with the threaded bore 42 in plate 38 until insert shoulder 48 engages the surface 39 of plate 38. The insert shoulder is of such dimension as to be accommodated within the counterbore in plate 40. Plate 40 is placed in adjacent position to plate 39 such that surface 41 is adjacent to surface 39. A threaded fastener 54, having a head 55, is passed through a washer 52, passed through aperture 51 in plate 40, and threadably engaged with internal threads 47 in the insert, thereby securing the two plates to form a dynamically loaded structural joint.

[0014] FIG. 3 is another embodiment of the mounting apparatus. In this embodiment, mounting apparatus 56 comprises two plates 58 and 60, held in clamping relationship by fastener 61. Fastener 61, which may have threads 57, may be passed through washer 65. Plate 58 has an aperture 66 through which the fastener 61 may be inserted. Plate 60 is equipped with a threaded blind bore 62, into which may be threaded insert 64. The insert 64 has a threaded aperture 63 into which the fastener 61 may be cooperatively threaded. Insert 64 has shoulders 67 and 68 which, when the insert is fully threaded into the bore, surface 70 on the shoulders may be (but not necessarily) brought into engagement with surface 72 of plate 60. The shoulders 67 and 68 are dimensioned such that together they form a diameter or dimension 74, which is accommodated within offset counterbore 76 in plate 58. While FIG. 5 describes the counterbore in plate 58, it is understood that such a counterbore may be in both plates. As seen in FIG. 5, counterbore 76 has a first end 78 with a first diameter or dimension 80 separated by a length L from a second end 82, with a second diameter or dimension 84. Diameter or dimension 80 is greater than size than



diameter or dimension **84**. Diameter or dimension **80** is of a sufficient size to accommodate the diameter or dimension **74** of threaded insert **64**. In this regard, the counterbore may be oblong. In other embodiments, it is not necessary to have first and second diameters or dimensions in the counterbore **76**. In some embodiments, it is possible to have a counterbore that has angular dimensions, such that the first end is of sufficient size to accommodate the diameter or dimension **74**, and is of a larger dimension than the second end. When the apparatus of FIGS. **3** and **5** is subjected to shear forces **88**, plate **58** moves in direction **90**. As plate **58** moves in direction **90**, the offset counterbore moves into engagement with the diameter or dimension **74** of the threaded insert **64**. As the force moves the counterbore wall **125** into engagement with the insert diameter or dimension, shear force is dissipated and its energy is absorbed. Material from plate **58** may be deformed in the process to absorb the interaction of the threaded diameter or dimension and the counterbore wall **125** without causing the mounting apparatus to come apart, thereby providing for a dynamically mounted joint. The embodiment of FIG. **3** may result in different clamping arrangements between the plates and the insert and the fastener. One possibility is that, with the insert fully threaded into the bore, and fastener plate **58** clamps with plate **60**, and surfaces **70** and **72** are engaged with each other, but surface **70** does not clamp with insert **68**. In another embodiment, plate **58** clamps with plate **60** and surfaces **70** and **72** are in engagement with each other. In yet another embodiment, plate **58** and **60** are clamped together, and the threaded insert may engage the counterbore surface **59**.

[0015] FIG. **4** is another embodiment of the mounting structure of FIGS. **3** and **5**, wherein plate **92** has a counterbore **94** to accommodate the insert **96**. In this embodiment, the insert, which may be threaded, engages both the counterbore **94** of plate **92**, but also engages counterbore **98** of plate **100** when the apparatus is subjected to dynamic forces as described with reference to FIG. **3** and FIG. **5**. Thus the insert as shown in FIG. **4** does not experience stress riser issues at the insert shoulder, and when assembled, the mounting apparatus couples plates **100** to **92** through the insert **96** when the fastener **49** is passed through aperture **17** and threaded into the insert as at **19**. It is understood that in this embodiment, both plate **92** and **100** may have a counterbore similar to that as described in FIG. **5**. Those skilled in the art also appreciate that the tapered slot as described in the previous figures may both be rounded counterbores in plates **92** and **100**.

[0016] FIG. **6** is another view of a dynamically loaded joint, showing a side view of the interaction of the threaded insert diameter and a pocket. Specifically, plate **102** has an aperture **104** into which is placed insert **106**. Insert **106** has shoulders **108** and **110**, with surfaces **112** and **114** which, when the insert is placed into the aperture, engage surface **116** on plate **102**. Those skilled in the art understand that both shoulders **108** and **110** are formed by the diameter **115** of the insert **106**, and are called out as separate numbers merely for clarity. Similarly, the surfaces depicted as **112** and **114** are actually the same surface in a top view, but are called by separate reference numbers in this side view. The shoulders form an insert diameter or dimension **115**, which is accommodated by a first diameter or dimension **117** in pocket **118**. The pocket **118** has a second diameter or dimension **120** separated from the first diameter or dimension by a length **122**. The second diameter or dimension is

smaller than the first diameter or dimension. In this regard, the structure as described is also seen in FIG. **5**. Motion between the two plates **102** and **124** causes the diameter or dimension of the insert to contact and interact with the side walls **125** (see FIG. **5**) in the pocket until the motion of one plate relative to the other is stopped by interaction of the insert with the sides of the pocket. As the one plate moves in direction **90** in relation to the other, it encounters shear forces **88** (as shown in FIG. **5**) until the energy is absorbed, thereby relieving the motion between the plates without separating the dynamically loaded structure.

[0017] Any of the described mounting structures may be used for joining dynamically loaded structures. One piece of structure is attached to another piece of structure with a bolt and an insert that is mechanically loaded as a result of translational movement of the two pieces of structure. The insert may be threaded or otherwise joined, such as a physical/chemical bonding, or a press fit, or a friction weld) into the main piece of structure, and, may have internal threads for attachment and a shoulder feature that sits on the surface of the one of the structures. The other piece of the structure (mating structure) has a counterbore that fits over the shoulder of the insert that transfers load when translation occurs between the two structures (members) to be joined. A fastener, such as a bolt, threads into the internal thread of the insert holding two pieces of structure together. In one embodiment, the configuration of the counterbore depth is tolerance such that the clamp load provided by the bolt clamps the two members together. The counterbore may be oblong or circular. An oblong counterbore allows for the members to be joined at an angle and will load the fastener and counterbore in three directions and the fastener in one direction. The mounting structures as described may be tuned as one might tune a shear pin, so that it could be constructed to take certain forces without failure but fail if those forces are exceeded.

[0018] The tolerances of the fastener, insert major diameter and counterbore diameter can be utilized to adjust the load sharing between the fastener (bolt) and the insert. The fastener can be utilized to adjust the load sharing between the fastener and the insert. The insert can be comprised of different materials dependant on the properties required. To maximize the energy the joint can transmit, a fastener with superb elongation and energy to failure properties may be selected.

[0019] The mounting structures as described may be suited for structural joints and may be selected for use in the application of under armor to eliminate the need for a puck on the inside face of the structure. The insert is threaded into the structure and can be replaced without reworking the base structure. The mounting apparatus as described can be utilized to reduce weight by allowing for the use of smaller fasteners (bolts). In addition the shear strength and tensile strength of the joint can be tuned independently by varying the sizes of the engaged counterbore and the fastener (bolt). Joints may be designed to have different shear strength in different directions by shaping of the counterbore.

[0020] Although the steps of the above-described processes have been exemplified as occurring in a certain sequence, such processes could be practiced with the steps performed in a different order. It should also be understood that certain steps could be performed simultaneously, that other steps could be added, or that certain steps could be



omitted. In other words, the descriptions of the processes are provided for the purpose of illustration, and should not limit the claimed invention.

[0021] Accordingly, it is to be understood that the above description is intended to be illustrative and not restrictive. Many embodiments and applications other than the examples provided would be apparent upon reading the disclosure. The scope of the invention should be determined with reference to the appended claims along with the full scope of equivalents. It is intended that future developments will occur, and that embodiments of the disclosed systems and methods will incorporate and be incorporated with such future developments.

[0022] Use of singular articles such as “a,” “the,” “said” together with an element means one or more of the element unless a claim expressly recites to the contrary.

What is claimed as new and desired to be protected by Letters Patent of the United States is:

1. A mounting structure apparatus, comprising:  
A first member having an aperture through a first surface and communicating with a counterbore in a second surface;  
a second member having a blind bore coaxial with said first member aperture;  
an insert engageable with said counter bore, and  
a fastener insertable into said first member aperture and interactive with said second member blind bore to join said first member to said second member.
2. The mounting apparatus of claim 1, wherein said insert is a projection from a surface of the second member inter-

active with said counter bore in said first member, said insert having internal threads interactive with threads on said fastener.

3. The mounting apparatus of claim 1, wherein said insert is equipped with internal threads and external threads, said external insert threads engageable with threads in a blind aperture in said second member, and said blind bore is threaded to engage said fastener; said insert further including shoulder portions to engage said first member counterbore.

4. The mounting apparatus of claim 1, wherein said counterbore is substantially circular.

5. The mounting apparatus of claim 1, wherein said counterbore is oblong.

6. The mounting apparatus of claim 1, wherein said counterbore has a first dimension to accommodate said insert, and a second dimension smaller than said first dimension; said second dimension separated from said first dimension by a length L; said counterbore tapered from said first dimension to said second dimension.

7. The mounting apparatus of claim 1, wherein said second member has a counterbore complimentary with said blind bore; said insert engageable with said second member counterbore and said first member counterbore.

8. The mounting apparatus of claim 7, wherein said second member has a counterbore concentric with said blind bore.

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